## Design Study

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## 1. Conceptual design and operation

Three probes cover the entire $\varnothing 120^{\prime \prime}(87.24 \mathrm{~mm})$ Field of View.
Each probe is mounted on a 2 degrees of freedom articulated arm composed of a crank arm and a lever arm, driven by 2 corresponding rotation motors: The crank and lever motors. (See Fig. 2)

Any position in the OSM field of view can be acquired by calculating appropriate values for theta and phi, noting that two possible solutions could be found due to symmetry.

The crank motor is secured to the Sensor and rotates the crank arm, precisely about the rotation axis of the crank motor referred to as the theta axis. The lever arm motor provides the necessary second degree of freedom by rotating the lever arm and all associated optics, about the phi axis.

Each probes are a at a different distance from the Focal plane, 15 mm apart, with the closest probe 5 mm upstream from the focal plane. This design allow each probe to freely roam the entire field without risk of colliding into an other probe. (See Fig. 1)

### 1.2 Field of View:

The Field of View at the Focal Plane is $\emptyset 5^{\prime \prime}$ ( 3.635 mm )
The Field of View at the probe is defined by it's distance from the Focal Plane / f \# (See Fig. 2)



### 1.1 Basic Design requirements:

Mechanism Type: $\quad \phi / \theta$
Patrolled Field: $\quad \varnothing$ 120" $(87.24 \mathrm{~mm})$
Focal Plane FoV : $\quad \varnothing 5^{\prime \prime}(3.635 \mathrm{~mm})$
Acquisition accuracy: 40 mas ( $30 \mu \mathrm{~m}$ )
Stability: $\quad 5 \mathrm{mas} / 3600 \mathrm{~s}(1 \mu \mathrm{~m})$
Position knowledge: $\quad<1 \mu \mathrm{~m}(\mathrm{TBC})$
Minimum Incremental motion: TBD
Operating Temperature: $-10^{\circ} \mathrm{C}+/-0.3$

## Position Accuracy

Total Position Accuracy of $30 \mu \mathrm{~m}$ at 300 mm arm extended length requires a crank and Lever rotation accuracy of: $\sin \alpha=30 \mu \mathrm{~m} / 300 \mathrm{~mm}=.03 / 300=.0001 \rightarrow \alpha=0.0057^{\circ}=.0057 \pi / 180=.0001 \mathrm{rad}$
The 250 mm lever arm motor is $86 \%$ longer than the 40 mm Crank arm
Crank motor rotation accuracy: $0.0057^{\circ} \times 86 \%=0.0049^{\circ}$
Lever motor rotation accuracy: $0.0057^{\circ} \times 14 \%=0.0008^{\circ}$

## 2. Optical Equation



Optical Layout (Fig. 3) is optimized when the following equations are verified:
2.1) $a=b \& c=d$
2.2) $a+b=x(c+d)$

Keeping the AO Focus away from the Probe mirror (FM1) gives:
2.3) $a=a_{1}+a_{2}$

Keeping each Lever arms on a different plane to avoid collision between each other gives a different value of a1 for each OSM The Lever Arm Length previously determined gives:
2.4) Lever Arm length $=\mathrm{b}+\mathrm{a}_{2}$

Replacing 3.3 \& 3.4 in 3.1 gives: $\mathrm{a}=\mathrm{b} \rightarrow \mathrm{a}_{1}+\mathrm{a}_{2}=$ Lever Arm length $-\mathrm{a}_{2}$
Solving for $\mathrm{a}_{2} \boldsymbol{\rightarrow} 2 \mathrm{a}_{2}=$ Lever Arm length $-\mathrm{a}_{1} \boldsymbol{\rightarrow} \mathrm{a}_{\mathbf{2}}=\left(\right.$ Lever Arm length $\left.-\mathrm{a}_{1}\right) / \mathbf{2}$
2.5) With $\mathrm{a}=\mathrm{b}=120$

| OSM\# | $\mathrm{a}_{1}$ | $\mathrm{a}_{2}=\mathrm{a}-\mathrm{a}_{1}$ | $\mathrm{a}=\mathrm{b}=\mathrm{a}_{1}+\mathrm{a}_{2}$ | Arm $=\mathrm{b}+\mathrm{a}_{2}$ |
| :---: | :---: | :---: | :---: | :---: |
| I | 10 | 110 | 120 | 230 |
| II | -5 | 125 | 120 | 245 |
| III | -20 | 140 | 120 | 260 |

## 3) Sizing the Probe

3.1) Probe FoV size depends directly from it's distance from the Focal plane.

The probe Fold Mirror intercept the light beam at a 45 degree angle creating an elliptical projection at a distance a1 from the Focal plane.
The minimum diameter of the mirror needs to be larger than the Ellipse Major Diameter.
The Larger Fold mirror will be at the furthest distance from the Focal plane OSM \#1 (a1 = -20)
Each Probe FoV is a potential vignetting of an other probe.
3.1.1) Largest FoV at the Fold Mirror \#1 (FM1) is at OSM \#1 (a1 = -20) Probe

$$
\begin{aligned}
& \text { Ellipse Minor Diameter: } \\
& \begin{aligned}
\mathrm{d} & =\mathrm{d} \text { at Focus Plan + (a1 / f\#) } \\
& =5 \times 0.727+(20 / 13.66) \\
& =3.635+1.464 \\
d & =5.10 \mathrm{~mm}
\end{aligned}
\end{aligned}
$$

Ellipse Major Diameter:

$$
D=d \sqrt{ } 2=5.1 \sqrt{ } 2
$$

$$
\mathrm{D}=7.21 \mathrm{~mm}
$$


3.1.2) Smallest FoV at the Fold Mirror \#1 (FM1) is at OSM \#2 (a1 = -5) Probe

$$
\begin{aligned}
& \text { Ellipse Minor Diameter: } \\
& \begin{aligned}
\mathrm{d} & =\mathrm{d} \text { at Focus Plan }+(\mathrm{a} 1 / \mathrm{f} \mathrm{\#}) \\
& =5 \times 0.727+(5 / 13.66) \\
& =3.635+.366 \\
\mathrm{~d} & =4.00 \mathrm{~mm}
\end{aligned}
\end{aligned}
$$

$$
\begin{aligned}
& \text { Ellipse Major Diameter: } \\
& D=d \sqrt{ } 2=4 \sqrt{ } 2 \\
& D=5.66 \mathrm{~mm}
\end{aligned}
$$

3.1.3) Medium FoV at the Fold Mirror \#1 (FM1) is at OSM \#3 (a1 = 10) Probe

$$
\begin{aligned}
& \text { Ellipse Minor Diameter: } \\
& \begin{aligned}
\mathrm{d} & =\mathrm{d} \text { at Focus Plan }+(\mathrm{a} 1 / \mathrm{f} \#) \\
& =5 \mathrm{X} 0.727+(10 / 13.66) \\
& =3.635+.732 \\
\mathrm{~d} & =4.37 \mathrm{~mm}
\end{aligned}
\end{aligned}
$$

```
Ellipse Major Diameter:
D = d \sqrt{ }{2}=4.37\sqrt{}{}2
D=6.18 mm
```


## 4) Probe Analysis

4.1) Design Features Common to all Probes


- Probe Cross Section

- Fold Mirror


## 4.2) Frequency Analysis

OSM \#1



OSM \#3

4.3) Probe Deflection, at rest, under it's own weight:

- Deflection analyzed on the longest Inclined probe (OSM \#2)



Probe Deflection

- Mass of the probe $w=m g=0.01 \mathrm{Kg} \times 9.81 \mathrm{~ms}^{-2}=0.098 \mathrm{~N}$
-Moment of Inertia: $\mathrm{I}=\mathrm{bh}^{3} / 12=4.57 \times 7.64^{3} / 12=169.8 \mathrm{~mm}^{4}$
-6061-T6 Module of Elasticity: $E=68,800 \mathrm{~N} / \mathrm{mm}^{2}$
-Max Deflection: $v=w L^{3} / 8 E I$
$v=0.098 \times 100^{3} /(8 \times 68,800 \times 169.8)$
$v=98000 / 93457920$


Probe Length
$v=0.001 \mathrm{~mm}$

## 5) Mass and CG

## 5.1) Lever Arm Assembly

Total Mass: 1,625 grams, CG located at the Axis of rotation, 41 mm above the Lever motor Interface.


## 5.2) OSM \#1 Assembly

Total Mass: 5,920 grams, CG located at the Crank Axis of rotation, 58 mm above the Lever motor Interface.


## 6) Lever Motor: Newport Stage URS75

## Newport URS Series Precision Rotation Stages

Dimensions
Dimensions in inches (millimeters)
MODEL URS 100B
 NOTE: ON URSISOB VERSIONS THE POSITION OF THESE 4 HOL ES IS IURNED OF $45^{\circ}$ ON USABLE DEPTH: K

MODEL SHOWN: URS IOOBPP


BOTTOM VIEW OF THE URSIOOBCC


Notes:
"The drive box of the URS75BCC exceeds. 20 in. 15 mm ) from the body
${ }^{3}$ URS 150B:-4 slots counterbered

| Design Details |
| :--- |
| Base Material Hardened steel with aluminum body <br> Bearings  <br> Drive <br> Mechanism Ground worm gear with self-compensating preload. Additional $1: 2.75$ drive belt with <br> URS-CC versions (no belt on URS-PP versions) <br> Worm Gear <br> Ratio $1: 90$ <br> Feedback CC: Worm mounted rotary encoder, 8,000 ctsirev, index pulse. <br> PP: None <br> Limit Switches Two independently adjustable optical limit switches <br> Origin Optical, fixed at position $0^{\circ}$. Typical $0.0005^{\circ}$ repeatability for URS-CC and $0.04^{\circ}$ repeatability <br> for URS-PP <br> Manual <br> Adjustment Via allen wrench at the end of the worm screw. Allen wrench is included. <br> Motor CC: UE $34 C C$ DC servo motor <br> PP: UE $34 P P$ Two phase stepper motor, 1 full step $=0.02^{\circ}$ <br> Cable  |


|  | PP |  | CC |  |
| :---: | :---: | :---: | :---: | :---: |
|  | Typical | Guaranteed | Typical | Guaranteed |
| Travel Range (') | 360 continuous $^{(1)}$ |  |  |  |
| Resolution (\%) | 0.0002 |  | 0.0005 |  |
| Minimum Incremental Motion (\%) | 0.0002 |  | 0.002 |  |
| Uni-directional Repeatability ( ${ }^{(0)}$ | 0.001 | 0.002 | 0.001 | 0.002 |
| Reversal Value (Hysteresis) (\%) | 0.006 | 0.01 | 0.002 | 0.004 |
| Absolute Accuracy ( ${ }^{\circ}$ ) | 0.016 | 0.030 | 0.012 | 0.023 |
| Maximum Speed ( ${ }^{\circ} / \mathrm{s}$ ) | 40 |  | 80 |  |
| Wobble (jurad) | 20 | 50 | 20 | 50 |
| Eccentricity ( $\mu \mathrm{m}$ ) |  | 3 |  | 3 |
| MTEF | 20,000 h at $25 \%$ load and with a $30 \%$ duty cycle |  |  |  |

1) With disabled lirnit switches
2) Equal to $1 / 100$ of a full step

See the Motion Control Metrology Primer section (see Motion Control Metrology Primer) for more information on typical and guara

Absolute Probe Position Accuracy on Field: $250 \mathrm{X} \tan 0.023=0.1 \mathrm{~mm}$
6.1) Load (Qh):

$$
\mathrm{Qh}=1.625 \mathrm{Kg}=15.9 \mathrm{~N}
$$

6.2) Cantilever Torque:

Distance between Lever Motor and center of Mass: $D=41 \mathrm{~mm}$

Cantilever Torque at Lever motor: $0.041 \mathrm{~m} \times 15.9=0.65 \mathrm{Nm}$
Off Center Load Requirement: $\mathrm{Q}<200 / 2 /(1+41 / 25)=38 \mathrm{~N} \rightarrow 16 \mathrm{~N}<38 \mathrm{~N}$

$$
16 \mathrm{~N}<38 \mathrm{~N} \rightarrow \mathrm{FS}=2.4
$$

Tip deflection angle $K($ urad $)=30 \times 0.65=19.6$ urad:


| Component | Manufacturer | P/N | Weight (Kg) | Mass (N) |
| :---: | :---: | :---: | :---: | :---: |
| T Mirror | PI | S-334 | 0.065 |  |
| Lens Holder | Newport | LPV-1 | 0.001 |  |
| Microswitch | Honeywell | 8SL125 | 0.476 |  |
| Lever Arm | Caltech | XXX | 1.083 |  |
| Lever Arm Assy |  |  | 1.625 |  |
| Total Mass |  |  | 1.625 | 15.94 |
| D (mm) | 41 |  |  |  |
| Cz (N) | 200 |  |  |  |
| a (mm) | 25 |  |  |  |
| Max Off center load (N) |  |  |  | 76 |
| K(urad/Nm) | 30 |  |  |  |
| Torque (Nm) | 0.65 |  |  |  |
| Deflection (urad) (rad) | 19.61 | $1.96077 \mathrm{E}-05$ |  |  |
| Proj Dist (mm) | 112 |  |  |  |
| Twisting of the tip (mm) | 0.0022 |  |  |  |
| Tip Horiz disp (mm) | 0.0022 |  |  |  |
| Tip Vert disp (mm) | 0.0000 |  |  |  |
|  |  |  |  |  |
|  |  |  |  |  |
|  |  |  |  |  |
|  |  |  |  |  |

## 7) Crank Motor: Newport Stage RV Series

## Specifications

| Travel Range ( ${ }^{\text {) }}$ | 360 continuous | With disabled limits, except HAT \& HAHLT |
| :---: | :---: | :---: |
|  | $\pm 170$ | HAT \& HAHLT versions |
| Resolution ( ${ }^{\text {a }}$ ) | 0.001 | Except RVS80, HAT \& HAHLT versions |
|  | 0.00025 | RVS80CC |
|  | 0.0001 | RVS80PP, equals $1 / 100$ of a full step |
|  | 0.0001 | RV120HAT \& HAHLT |
|  | 0.00075 | RV160HAT \& HAHLT |
|  | 0.0005 | RV240HAT \& HAHLT |
|  | 0.00035 | RV350HAT \& HAHLT |
| Minimum Incremental Motion ( ${ }^{\circ}$ ) | 0.001 | Except RVS80, HAT \& HAHLT versions |
|  | 0.00025 | RVS80CC |
|  | 0.0002 | RVS80PP |
|  | 0.0001 | HAT \& HAHLT versions |
| Uni-directional Repeatability ( ${ }^{\circ}$ ) | 0.001 typical, 0.002 guaranteed | Except RVS80CC, HAT \& HAHLT versions |
|  | 0.001 guaranteed | RVS80CC |
|  | 0.002 guaranteed | RVS80PP |
|  | 0.00011 typical, 0.0002 guaranteed | HAT \& HAHLT versions |
| Reversal Value (Hysteresis) ( ${ }^{\circ}$ ) | 0.001 typical, 0.002 guaranteed | Except RVS80PP, HAT \& HAHLT versions |
|  | 0.005 guaranteed | RVS80PP |
|  | 0.0006 typical, 0.001 guaranteed | HAT \& HAHLT versions |
| Absolute Accuracy ( ${ }^{\circ}$ ) | 0.007 typical, 0.01 guaranteed | Except RVS80, HAT \& HAHLT versions |
|  | 0.015 guaranteed | RVS80CC |
|  | 0.02 guaranteed | RVS80PP |
|  | 0.003 typical, 0.005 guaranteed | HAT \& HAHLT versions |
| Maximum Speed ( $\%$ s) | 80 | CC \& HAT motor option, except RVS80CC |
|  | 40 | RVS80CC |
|  | 20 | PP motor option |
|  | 16 | CCHL \& HAHLT motor option |
|  | 2 | PE motor option |
| Wobble ( $\mu \mathrm{rad}$ ) | 40 guaranteed | RVS80 |
|  | 10 typical, 20 guaranteed | RV120 \& RV160 |
|  | 8 typical, 16 guaranteed | RV 240 \& RV350 |
| Eccentricity ( $\mu \mathrm{m}$ ) | 1.4 typical, 4 guaranteed |  |
| MTBF (h) | 20,000 |  |

## Absolute Probe Position Accuracy on Field:

RVS80PP: $290 X \tan 0.02=0.101 \mathrm{~mm} \quad$ (\$3983)

RVS80CC: $290 \mathrm{X} \tan 0.015=0.076 \mathrm{~mm} \quad$ (\$4454)
RVS120HAT: $290 \mathrm{X} \tan 0.005=0.025 \mathrm{~mm} \quad$ (\$8920)
7.1) RVS80 Load $\mathrm{Qv}=5.920 \mathrm{Kg}=59.2 \mathrm{~N}$

Distance between Crank Motor and center of Mass: $D=58+32=90 \mathrm{~mm}$

Cantilever Torque at Crank motor: 0.090 m X 59N $=5.3 \mathrm{Nm}$

Max Off Center Load: 900/2 / (1 + 90/30) = 112 N

$$
59 \mathrm{~N}<112 \mathrm{~N} \rightarrow \mathrm{FS}=1.9
$$

Load Characteristics

| Load Characteristics | RVS80 | RV120 | RV160 | RV240 | RV350 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Cz, normal centered load capacity ( N ) | 900 | 1800 | 2700 | 4000 | 6500 |
| a, construction parameter (mm) | 30 | 40 | 50 | 70 | 100 |
| $\mathrm{b}^{*}$, (mm) except HAT \& HAHLT | 39 | 53 | 57 | 59 | 73 |
| $\mathrm{b}^{*}$, (mm) for HAT \& HAHLT |  | 71 | 75 | 77 | 91 |
| k, radial compliance ( $\mu \mathrm{rad} / \mathrm{Nm}$ ) | 3.5 | 1.5 | 0.6 | 0.3 | 0.1 |
| $\mathrm{Q}_{\text {H, }}$, Off-center load, vertical rotation axis | $\mathrm{Q}_{\mathrm{H}} \mathrm{Cz} /(1+\mathrm{D} / \mathrm{a})$ |  |  |  |  |
| $\mathrm{Q}_{\mathrm{V} \text {, Off-center load, horizontal rotation axis }}$ | $\mathrm{Q}_{\mathrm{V}} \mathrm{Cz} / 2 /$ (1+D/a) |  |  |  |  |

* Construction parameter $=$ Distance between the top surface of the RV stage and the bearing center

Example:
$\mathrm{Q}_{\mathrm{V}}$ at a distance of 80 mm from the top surface for a RV160HAT rotation stage, $\mathrm{D}=80 \mathrm{~mm}+75 \mathrm{~mm}=155 \mathrm{~mm}$ :
$\mathrm{Q}_{\mathrm{V}}=2700 \mathrm{~N} / 2 /(1+155 \mathrm{~mm} / 50 \mathrm{~mm})=329 \mathrm{~N}$

## Load Rotation Axes




## 6) Absolute Probe Position Accuracy on Field

- Lever Motor Newport Stage URS75: $250 \mathrm{X} \tan 0.023=0.1 \mathrm{~mm}$
- Crank Motor Newport Stage RVS80PP: 290 X tan $0.02=0.101 \mathrm{~mm}$
- Crank Motor Newport Stage RVS80CC: 290 X $\tan 0.015=0.076 \mathrm{~mm}$
- Crank Motor Newport Stage RVS120HAT: 290 X tan $0.005=0.025 \mathrm{~mm}$

NGAO VIEW


## OPTICAL PATH

$$
a=b=120 \mathrm{~mm} \quad \& \quad c=d=240 \mathrm{~mm}
$$



## OPEN ISSUES

1) The entire 200 Kg LOWFS OSM should be installed on linear stages to provide the 5 mm focus adjustment necessary to compensate for the Dichroic pickoff motion. (Accuracy TBD)
2) The 1 mm sag caused by the curved focus around the Probes FM1 is not solved yet.


NGAO VIEW


